## Fully-developed, pressure-driven flow of an incompressible, isothermal fluid through a straight open duct with rectangular cross-section: data from DNS

## Description of the flow

We are considering the flow of an incompressible and isothermal fluid in a straight open duct with rectangular cross-section of duct full-height $h$ (cf. figure 1). The flow field is assumed to be streamwise periodic over a period of length $L_{x}$ and a constant flow rate is imposed at each time step.

## Flow parameters

The problem is governed by two parameters, namely: the bulk Reynolds number $R e_{b}=u_{b} h / \nu$, where $u_{b}$ is the bulk velocity and $\nu$ the kinematic viscosity; and aspect ratio $A=L_{z} /(2 h)$. Note the factor of 2 difference between the above aspect ratio definition and more conventional width-toheight ratio. The current definition is used to maintain the consistency between the open and the closed duct with the identifical hydraulic diameter ( $D_{H}=4 \mathcal{A} / P$, where $\mathcal{A}$ is duct cross-sectional area and $P$ is wetted perimeter) having the same aspect ratio. Table 1 shows the simulated Reynolds numbers as well as aspect ratios.

## Numerical method and resolution

- Incremental-pressure projection method;
- Crank-Nicholson scheme for the viscous terms;
- Three-step low-storage Runge-Kutta method for the non-linear terms [1];
- Truncated Fourier series in the streamwise direction ( $2 / 3$ de-aliasing),
- Chebyshev polynomials in the cross-stream (collocated grid);


## Numerical parameters

The data included in this repository is characterized by the following features:

- Domain size: $L_{x}=8 \pi h$


Figure 1: Coordinate system and geometry of open duct simulations. Shaded planar boundaries represent no-slip walls, whereas the transparent plane enclosed by dotted lines represents free-slip boundary.

- Time step: $C F L \leq 0.3$
- Streamwise grid spacing: $\Delta x^{+} \leq 15.2$
- Maximum cross-stream grid-spacing: $\max \left(\Delta y^{+}, \Delta z^{+}\right) \leq 4.0$
- Statistics accumulated over time: $t_{\text {stat }} \geq 6009$


## Available data

The data-set contains the following data items:

- components of the time-averaged velocity vector $\langle\boldsymbol{u}\rangle(y, z)$;
- components of the Reynolds stress tensor $\left\langle u_{i}^{\prime} u_{j}^{\prime}\right\rangle(y, z)$, where the fluctuation is defined as $\boldsymbol{u}^{\prime}=\boldsymbol{u}-\langle\boldsymbol{u}\rangle$.


## Data format and location

Data is presented in the form of binary files. A script for reading the data and plotting it with Matlab (or GNU/octave) is provided. The data is located below the following URL:
http://www.ifh.kit.edu/dns_data/duct/OPEN_duct/

## Contact

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## References

[1] R Verzicco and P Orlandi. A finite-difference scheme for three-dimensional incompressible flows in cylindrical coordinates. J. Comput. Phys., 123(2):402-414, 1996.

| $A$ | $R e_{b}$ | $R e_{\tau}$ | $N_{x} \times N_{y} \times N_{z}$ | $\Delta x^{+}$ | $\max \left\{\Delta y^{+}, \Delta z^{+}\right\}$ | $t_{\text {stat }} u_{b} / h$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.5 | 1900 | 124 | $256 \times 97 \times 97$ | 12.2 | 2.0 | 31083 |
| 0.5 | 2000 | 142 | $256 \times 97 \times 97$ | 13.9 | 2.3 | 20323 |
| 0.5 | 2205 | 154 | $256 \times 97 \times 97$ | 15.2 | 2.5 | 6456 |
| 0.5 | 3000 | 203 | $384 \times 97 \times 97$ | 13.3 | 3.3 | 8301 |
| 0.5 | 5000 | 325 | $512 \times 129 \times 129$ | 16 | 4.0 | 12021 |
| 1 | 1450 | 102 | $256 \times 97 \times 193$ | 10.2 | 1.69 | 9183 |
| 1 | 1480 | 103 | $256 \times 97 \times 193$ | 10.1 | 1.68 | 5248 |
| 1 | 1500 | 104 | $256 \times 97 \times 193$ | 10.2 | 1.69 | 11807 |
| 1 | 1550 | 107 | $256 \times 97 \times 193$ | 10.4 | 1.74 | 9180 |
| 1 | 1800 | 125 | $256 \times 97 \times 193$ | 12.3 | 2.05 | 11449 |
| 1 | 2205 | 150 | $256 \times 97 \times 193$ | 14.8 | 2.46 | 12940 |
| 1 | 3000 | 197 | $384 \times 97 \times 193$ | 12.9 | 3.23 | 9082 |
| 1 | 3500 | 226 | $384 \times 97 \times 193$ | 14.8 | 3.70 | 14826 |
| 1 | 4000 | 254 | $512 \times 129 \times 257$ | 12.5 | 3.12 | 9016 |
| 1 | 4500 | 282 | $512 \times 129 \times 257$ | 13.8 | 3.45 | 10304 |
| 1 | 5000 | 309 | $512 \times 129 \times 257$ | 15.2 | 3.80 | 8005 |
| 1 | 5500 | 336 | $768 \times 193 \times 385$ | 11.0 | 2.75 | 7156 |
| 1 | 6000 | 363 | $768 \times 193 \times 385$ | 11.9 | 3.00 | 9686 |
| 1 | 6500 | 388 | $768 \times 193 \times 385$ | 12.7 | 3.20 | 6009 |
| 1 | 7000 | 415 | $768 \times 193 \times 385$ | 13.6 | 3.40 | 6310 |
| 1.5 | 2205 | 147 | $256 \times 97 \times 289$ | 14.4 | 2.4 | 18166 |
| 2 | 1080 | 74 | $256 \times 97 \times 385$ | 7.3 | 1.2 | 8587 |
| 2 | 1100 | 75 | $256 \times 97 \times 385$ | 7.4 | 1.2 | 11449 |
| 2 | 1150 | 77 | $256 \times 97 \times 385$ | 7.6 | 1.3 | 11449 |
| 2 | 1200 | 81 | $256 \times 97 \times 385$ | 7.9 | 1.3 | 12403 |
| 2 | 1250 | 84 | $256 \times 97 \times 385$ | 8.3 | 1.4 | 12403 |
| 2 | 1300 | 89 | $256 \times 97 \times 385$ | 8.7 | 1.4 | 12403 |
| 2 | 1350 | 92 | $256 \times 97 \times 385$ | 9.0 | 1.5 | 8587 |
| 2 | 1400 | 95 | $256 \times 97 \times 385$ | 9.4 | 1.6 | 10494 |
| 2 | 1450 | 99 | $256 \times 97 \times 385$ | 9.7 | 1.6 | 8587 |
| 2 | 1500 | 102 | $256 \times 97 \times 385$ | 10.0 | 1.7 | 13357 |
| 2 | 1600 | 108 | $256 \times 97 \times 385$ | 10.6 | 1.8 | 8587 |
| 2 | 1800 | 121 | $256 \times 97 \times 385$ | 11.9 | 2.0 | 8587 |
| 2 | 2205 | 145 | $256 \times 97 \times 385$ | 14.2 | 2.4 | 23778 |
| 2 | 3000 | 191 | $384 \times 97 \times 385$ | 12.5 | 3.1 | 9817 |
| 2 | 5000 | 297 | $512 \times 129 \times 513$ | 14.6 | 3.7 | 6144 |
| 4 | 2205 | 143 | $256 \times 97 \times 769$ | 14.0 | 2.3 | 11270 |
| 8 | 1800 | 120 | $256 \times 97 \times 1153$ | 11.8 | 2.6 | 8953 |
| 8 | 2205 | 143 | $256 \times 97 \times 1153$ | 14.1 | 3.1 | 8140 |
|  |  |  |  |  |  |  |

Table 1: Simulation parameters.

